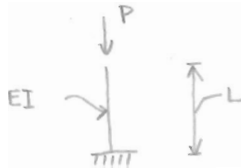


GOAL: Illustrate how P-Delta Forces are related to or used in buckling analysis.

MODEL:



Buckling analysis involves the solution of the generalized eigenvalue problem:

$$[K - \lambda G(v)] \Psi = 0$$

Ref.: CSI Analysis Reference Manual  
chapter "Analysis Cases",  
section "Linear Buckling Analysis"

$K$ ... stiffness matrix

$G(v)$ ... geometric stiffness matrix due to load vector  $v$

$\lambda$ ... unknown eigenvalues diagonal matrix

$\Psi$ ... matrix of corresponding eigenvectors

For our cantilever problem, the equation can be rewritten as follows:

$$[K_e + \lambda K_g] \Psi = 0$$

$\uparrow$  non-trivial solution  
 $\uparrow$  to obtain non-trivial solution,  
 this matrix must be singular

where

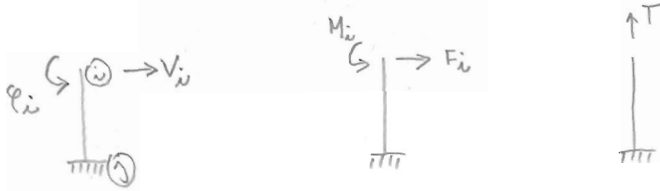
$$K_e = \frac{EI}{L^3} \begin{bmatrix} 12 & 6L \\ 6L & 4L^2 \end{bmatrix}$$

$$K_G = \frac{T}{30L} \begin{bmatrix} 36 & 3L \\ 3L & 4L^2 \end{bmatrix}$$

$$\text{note: } \begin{Bmatrix} F_i \\ M_i \end{Bmatrix} = K_e \begin{Bmatrix} v_i \\ \varphi_i \end{Bmatrix}$$

$$\text{note: } \begin{Bmatrix} F_i \\ M_i \end{Bmatrix} = K_G \begin{Bmatrix} v_i \\ \varphi_i \end{Bmatrix}$$

Edward L.  
Wilson: Static  
& Dynamic Analysis  
of structures, 2004,  
p. 120, 121



SIGN CONVENTION FOR  $v_i$ ,  $\varphi_i$ ,  $F_i$ ,  $M_i$  and  $T$

$$\text{Then } K_t = K_e + \lambda K_G = \frac{EI}{L^3} \begin{bmatrix} 12 & 6L \\ 6L & 4L^2 \end{bmatrix} + \lambda \frac{T}{30L} \begin{bmatrix} 36 & 3L \\ 3L & 4L^2 \end{bmatrix} =$$

$$\begin{bmatrix} \frac{12EI}{L^3} + \frac{6}{5} \frac{T}{L} \lambda & \frac{6EI}{L^2} + \frac{1}{10} T \lambda \\ \frac{6EI}{L^2} + \frac{1}{10} T \lambda & \frac{4EI}{L} + \frac{2}{15} T L \lambda \end{bmatrix}$$

To obtain nontrivial solution for the eigen value problem, the determinant of the  $K_t$  matrix must be equal to zero:

$$\det(K_t) = 0$$

$$\left( \frac{12EI}{L^3} + \frac{6}{5} \frac{T}{L} \lambda \right) \left( \frac{4EI}{L} + \frac{2}{15} T L \lambda \right) - \left( \frac{6EI}{L^2} + \frac{1}{10} T \lambda \right)^2 = 0 \quad \left| \text{solve for unknown } \lambda \right.$$

$$\frac{48(EI)^2}{L^4} + \frac{24}{15} \frac{EI}{L^2} T\lambda + \frac{24}{5} \frac{EI}{L^2} T\lambda + \frac{12}{75} T^2 \lambda^2 -$$

$$- \frac{36(EI)^2}{L^4} - 2 \cdot \frac{6}{10} \frac{EI}{L^2} T\lambda - \frac{1}{100} T^2 \lambda^2 = 0$$

$$12 \left( \frac{EI}{L^2} \right)^2 + \lambda \frac{EI}{L^2} T \left[ \frac{24}{15} + \frac{24}{5} - \frac{12}{10} \right] + \lambda^2 T^2 \left[ \frac{12}{75} - \frac{1}{100} \right] = 0$$

$$\frac{48 + 144 - 36}{30} = \frac{156}{30} = \frac{26}{5}$$

$$\frac{48 - 3}{300} = \frac{45}{300} = \frac{3}{20}$$

$$\underbrace{\frac{3}{20}}_a T^2 \lambda^2 + \underbrace{\frac{26}{5} \frac{EI}{L^2} T}_b \lambda + \underbrace{12 \left( \frac{EI}{L^2} \right)^2}_c = 0$$

$$ax^2 + bx + c = 0$$

$$x_{1,2} = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a}$$

solve this quadratic equation:

$$D = b^2 - 4ac = \frac{26^2}{5^2} \left( \frac{EI}{L^2} \right)^2 T^2 - 4 \left( \frac{3}{20} T^2 \right) \left( 12 \left( \frac{EI}{L^2} \right)^2 \right) = \left( \frac{EI}{L^2} \right)^2 T^2 \left( \frac{26^2}{5^2} - \frac{36}{5} \right)$$

$$= \left( \frac{EI}{L^2} \right)^2 T^2 \frac{676 - 180}{25} = \left( \frac{EI}{L^2} \right)^2 T^2 \frac{496}{25}$$

$$\sqrt{D} = \frac{EI}{L^2} T \frac{\sqrt{496}}{5} = \frac{4\sqrt{31}}{5} \frac{EI}{L^2} T$$

$$\lambda_{1,2} = \frac{-\frac{26}{5} \frac{EI}{L^2} T \pm \frac{4\sqrt{31}}{5} \frac{EI}{L^2} T}{2 \cdot \frac{3}{20} T^2} = \frac{\frac{EI}{L^2} T \left( -\frac{26}{5} \pm \frac{4\sqrt{31}}{5} \right)}{\frac{3}{10} T^2} = \frac{EI}{L^2 T} \cdot \frac{10}{3} \cdot \frac{-26 \pm 4\sqrt{31}}{5} =$$

$$= \frac{2}{3} (-26 \pm 4\sqrt{31}) \frac{EI}{L^2 T} = \frac{4}{3} (-13 \pm 2\sqrt{31}) \frac{EI}{L^2 T} = \begin{cases} -2.4860 \frac{EI}{L^2 T} \\ -32.1807 \frac{EI}{L^2 T} \end{cases}$$



When compressive force  $P$  is applied, then

$$T = -P \quad \text{and} \quad \lambda_1 = 2.4860 \frac{EI}{L^2 P} \quad (\text{lowest root})$$

Accounting for pre-existing P-Delta Forces in SAP2000:

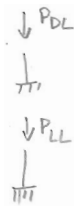
Load case 1: DL ... nonlinear with P-Delta effects

Load case 2: LL ... linear buckling using stiffness at the end of case 1



loads from case 1 are NOT included

in case 2, but P-Delta effects ARE included



Therefore  $P_{cr} = P_{DL} + \phi P_{LL}$

and reported

↑ buckling factor determined by SAP2000 for  $P_{LL}$ , taking into account the effect of  $P_{DL}$  already acting on the structure